

$\frac{d}{dt} \left(\int_{\Omega(t)} u^m dx + \int_{\partial\Omega(t)} v^m ds \right) = - \int_{\Omega(t)} m(u,v)^{\frac{m-1}{2}} (\nabla u \cdot \nabla v) dx$

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**"SPLIT ICE MAKING AND DELIVERY SYSTEM
FOR MARITIME AND OTHER APPLICATIONS"**

BACKGROUND OF THE INVENTION

1. Field of the Invention

5 The present invention relates to ice makers for
marine vessels and recreational vehicles (RV) and other
applications, and, more particularly, to a split marine
ice making and delivery system which locates the ice
making sub-assembly adjacent or in close proximity to the
10 ice storage bin and away from the condenser unit or
compressor unit.

2. General Background

Presently, refrigerant systems for marine
applications are made of a single unit which pushes ice
15 threw long tubes which frequently clog such as, when
pieces of ice adhere together. Moreover, such
refrigerant systems are relatively noisy as the ice is
pushed to remote locations 20, 30, and 40 of feet away.

For example, U.S. Patent No. 4,922,724, issued to
20 Grayson, et al., entitled "MARINE ICE MAKING AND DELIVERY
SYSTEM" discloses a refrigeration circuit located on the
engine deck of a marine craft having an ice making
assembly and a flexible conduit coupled to the output of
the ice making assembly. The flexible conduit has a

length sufficient to reach upper levels of the marine craft and reaches horizontally remote locations from the refrigeration circuit to deliver ice.

U.S. Patent Nos. 4,576,016 and 4,574,593, issued to
5 Nelson, entitled "ICE MAKING APPARATUS" discloses a combination evaporator and auger-type ice-forming assembly operatively disposed between an ice product receiving area and a drive means assembly.

U.S. Patent Nos. 4,433,559, issued to King-Seeley
10 Thermos Co., entitled "ICE MAKING APPARATUS" discloses an ice-making apparatus having a rotatable auger and a helical evaporator. The output of the ice-making apparatus is delivered to an extruder mechanism which causes flaked ice from the ice-making apparatus to be
15 compacted or compresses and formed into discrete ice bodies or cubes. The ice bodies or cubes are delivered to a storage bin via a conduit.

As can be appreciated there is a continuing need for a split ice making and delivery system which eliminates
20 forcing through very long conduits ice product which oftentimes becomes clogged.

As will be seen more fully below, the present invention is substantially different in structure, methodology and approach from that of the prior

refrigeration systems.

SUMMARY OF THE PRESENT INVENTION

The preferred embodiment of split ice making and delivery system of the present invention solves the
5 aforementioned problems in a straight forward and simple manner.

Broadly, the present invention contemplates a split ice making and delivery system comprising: a condenser and compressor sub-assembly which compresses and
10 condenses refrigerant; a remote ice making sub-assembly having a rotating auger, a fresh water freeze chamber adapted to be filled with portable fresh water and an outlet wherein rotation of said auger forces out, of said outlet, ice product; and, a refrigerant delivery sub-
15 assembly coupled to said condenser and compressor sub-assembly and said remote ice making sub-assembly for delivering therebetween said refrigerant wherein said refrigerant delivery sub-assembly has a length sufficient to reach a remote room or remote location and to reach
20 said remote ice making sub-assembly remote from said condenser and compressor sub-assembly.

In view of the above, an object of the present invention is to provide a split ice making and delivery system comprising an ice storage bin which is located in

close proximity to the remote ice making sub-assembly;
and, means for channeling ice product from the remote ice
making sub-assembly to the ice storage bin wherein the
ice channeling means has a length less than 10 feet.

5 Another object of the present invention is to
provide a split ice making and delivery system having a
remote ice making sub-assembly which is capable of
producing 380-500 pounds of ice per day.

10 A further object of the present invention is to
provide a split ice making and delivery system having a
combination remote ice making sub-assembly and ice
storage bin wherein the remote ice making sub-assembly
includes a compact housing for storing the remote ice
making sub-assembly wherein the housing has a height of
15 approximately 29 ½ inches and a width and depth of 12
inches.

20 A still further object of the present invention is
to provide a split ice making and delivery system having
a remote ice making assembly which includes an evaporator
coiled around an auger having a refrigerant inlet line
receiving refrigerant from via a refrigerant delivery
line of the refrigerant delivery sub-assembly from the
condenser and compressor sub-assembly to the refrigerant
inlet line and a refrigerant outlet line expels spent
25 refrigerant on return refrigerant delivery line to the

condenser and compressor sub-assembly.

A still further object of the present invention is to provide a split ice making and delivery system having a control temperature sensor integrated into or affixed to an ice storage bin wherein as the ice product reaches a predetermined level, a decrease in temperature is realized at the control temperature sensor and the condenser and compressor sub-assembly and the remote ice making sub-assembly are deactivated.

A still further object of the present invention is to provide a split ice making and delivery system having a thermo-expansion valve in-line between the remote ice making sub-assembly and the condenser and compressor sub-assembly.

In view of the above, a feature of the present invention is to provide a split ice making and delivery system which eliminates long conduits through which ice is channeled to a remote ice storage bin.

Another feature of the present invention is to provide a split ice making and delivery system which minimizes the operating noise.

A further feature of the present invention is to provide a split ice making and delivery system which channels through long conduits refrigerant to remote location in a marine vessel or craft or RV.

BRIEF DESCRIPTION OF THE DRAWING

For a further understanding of the nature and objects of the present invention, reference should be had to the following description taken in conjunction with the accompanying drawings in which like parts are given like reference numerals and, wherein:

FIGURE 1 illustrates a view of the split ice making and delivery system of the present invention deployed on a marine vessel;

FIGURE 2 illustrates a general schematic diagram of the refrigeration circuit of the split ice making and delivery system of the present invention;

FIGURE 3 illustrates a perspective view of the remote ice making sub-assembly in combination with an ice bin of the present invention;

FIGURE 4 illustrates a cross sectional view along the **PLANE 4 - 4** of **FIGURE 5**;

FIGURE 5 illustrates a perspective the internal components of the remote ice making sub-assembly; and,

FIGURE 6 illustrates a cross-sectional view along the **PLANE 6-6** of **FIGURE 3**.

DETAILED DESCRIPTION OF THE PREFERRED EMBODIMENT

Referring now to the drawings and in particular **FIGURES 2 - 5**, the split ice making and delivery system of the present invention is generally referenced by the numeral **10**. The split marine ice making and delivery system **10** is generally comprised of a compressor and condenser sub-assembly **20** in fluid communication with a remote ice making sub-assembly **50** via a refrigerant delivery sub-assembly **40**. The split marine ice making and delivery system **10** may further include an ice storage bin **70** or it may stand alone. The ice storage bin **70** includes a ice scoop or ladle **71**.

Referring now to **FIGURE 1**, the split marine ice making and delivery system **10** is adapted for marine applications wherein the refrigeration circuit **100** of the split ice making and delivery system **10** is split into two general sub-assemblies, the compressor and condenser sub-assembly **20** and the remote ice making sub-assembly **50** adapted to be separated by many feet, compartments or floors of a marine vessel **1** via a refrigerant delivery sub-assembly **40**. In an alternate embodiment, the split

marine ice making and delivery system 10 is adapted for RV applications.

In the marine application, the compressor and condenser sub-assembly 20 is adapted to be deployed in the engine room 2 where raw water or sea water is easily accessible while the remote ice making sub-assembly 50 is adapted to be located in another compartment or floor 3 remote from the engine room 2. Since, the remote ice making sub-assembly 50 is in fluid communication with the compressor and condenser sub-assembly 20 via the refrigerant delivery sub-assembly 40, the ice 90 does not have to be communicated remotely to the ice storage bin 70 on the marine vessel. Instead, the refrigerant fluid having a natural tendency to flow is easily communicated remotely in the refrigerant delivery sub-assembly 40 between the compressor and condenser sub-assembly 20 and the remote ice making sub-assembly 50. Hence, clogging ice in such long conduits is eliminated.

Additionally, locating the compressor and condenser sub-assembly 20 in the engine room 2 or other location minimizes the impact of the operational noise therefrom on the occupants of the marine vessel 1.

Referring now to **FIGURES 3 - 6**, the remote ice making sub-assembly **50** is housed in housing unit **52**. The remote ice making sub-assembly **50** and housing unit **52** are compact and are designed to be located in close proximity to the ice storage bin **70**. In the exemplary embodiment, the housing unit **52** has affixed thereto the ice storage bin **70**. As best seen in **FIGURE 6**, the housing unit **52** has mounted to a front surface thereof a first coupler or rail **73**. The ice storage bin **70** comprises a second coupler or channel guide **74** adapted to connect to or mate with the first coupler or rail **73** to secure the ice storage bin **70** to the housing unit **52**. The housing unit **52** further includes means for channeling ice **75** which is coupled to the chamber outlet **53b**. The means for channeling ice **75** includes any one of a hose or tubing having a length of a few inches up to 10 feet or a chute. The hose or tubing of the means for channeling ice **75** has a diameter of approximately 1 inch. Furthermore, the front of the housing unit **52** is provided with a manual reset button **78** to allow occupants to manually reset the system **10**.

In the preferred embodiment, the housing unit includes lid 76 and rear brackets 77 for affixing the housing unit 52 to a wall.

5 The remote ice making sub-assembly 50 includes a rotatable auger 54 rotatably mounted in a freeze chamber 66 and which is rotated by a high torque motor 56 connected via gear box 58 to the rotatable auger 54. Thereby, no other extruding mechanism is needed to force the ice through long conduits. The gear box 58 is
10 stacked above the high torque motor 56. The freeze chamber 66 and auger 54 are stacked above the gear box 58.

The remote ice making sub-assembly 50 further includes an evaporator 60 which is coiled around the
15 auger 54 and an insulating housing 64 encapsulating the evaporator 60. Refrigerant is supplied via the refrigerant delivery line 42a of the refrigerant delivery sub-assembly 40 from the compressor and condenser sub-assembly 20 to the refrigerant inlet line 61a of the
20 evaporator 60. The refrigerant outlet line 61b of the evaporator 60 expels the spent refrigerant on return

refrigerant delivery line **42b**. The return refrigerant delivery line **42b** delivers the spent refrigerant to the compressor and condenser sub-assembly **20**.

The auger **54** is selectively rotated by motor **56** to scrap or shave the frozen water in the freeze chamber **66** and create ice **90**. The fresh water from the fresh water reservoir **80**, which includes a float **83**, fills the freeze chamber **66**.

Referring again to **FIGURE 2**, the refrigeration circuit **100** will be described in more detail. The compressor and condenser sub-assembly **20** includes a compressor unit **22** which supplies a flowable gaseous refrigerant, such as refrigerant R-22, to the condenser unit **24** on the condenser refrigerant inlet line **25a**. The condenser unit **24** cools or liquefies the gaseous refrigerant and outputs, on the condenser's outlet line **25b**, the liquified refrigerant to the refrigerant inlet line **61a** of the evaporator **60** via the refrigerant delivery line **42a**. A TXV or thermo-expansion valve **45** for metering the refrigerant is coupled in-line between the refrigerant delivery line **42a** and the refrigerant

inlet line **61a** of the evaporator **60**.

The liquified refrigerant flows through the evaporator **60** and exits the evaporator at the refrigerant outlet line **61b** and flows back to the compressor unit **22** where the refrigerant loop begins. As the liquified refrigerant flows through the evaporator **60**, the water in the freeze chamber **66** freezes via heat transfer.

The evaporator **60** surrounding the exterior of the freeze chamber **66** causes the fresh water therein to freeze as the refrigerant flows therethrough. As the auger **54** rotates the frozen fresh water is shaved to create ice **90**. Moreover, as the auger **54** rotates, the shaved ice **90** is channeled upward to chamber outlet **53b** where ice **90** is expelled and stored in ice storage bin **70**.

In the exemplary embodiment, the condenser unit **24** includes a water cooled, cooper-plated tubing having a raw water inlet line **26a** and a raw water outlet line **26b**. The raw water inlet line **26a** receives raw water from the engine room or from outside the marine vessel **1**. There is a conventional water controller valve **29** in inlet line

26a for controlling water in-take flow. As the raw water flows through the condenser unit 24, the spent raw water exits therefrom through the raw water outlet line 26b. The flow of the raw water through the condenser unit 24 is controlled via pumping unit 30.

The raw water inlet line 26a is an outer annular tubing and has concentric therethrough the condenser's refrigerant line (not shown) terminating between the condenser refrigerant inlet line 25a and the condenser refrigerant outlet line 25b. The raw water intake is controlled by the water controller 29 in line 26a which is controlled by the pressure of the system 10 for maximum efficiency of the system 10.

The water controller 29 is used in the system 10 to accommodate for a range of raw water temperatures such as from 40 degrees to 95 degrees Fahrenheit. The condensing unit 24 also has low and high pressure control.

The refrigeration circuit 100 further includes a control temperature sensor 85 integrated into or affixed to the ice storage bin 70. Thereby, as the ice level increases in the ice storage bin 70, the ice 90 will

reach the sensor's level. The control temperature sensor 84 is temperature sensitive to the temperature of ice and coupled to thermostat 87. The control temperature sensor 85 deactivates the motor 56, the pump 30 and compressor unit 22 thereby deactivating the refrigeration circuit 100. In other words, the compressor and condenser sub-assembly 20 and the remote ice making sub-assembly 50 are deactivated.

Moreover, a water switch 82 is provided to maintain water pressure at a minimum of 10 psi. If the fresh water reaches below 10 psi, the system 10 will deactivate until the pressure reaches 10 psi. The system can be deactivated by providing a conventional safety switch or thermostat in bin 70.

In the exemplary embodiment, the voltage (V) is 230 V or 115 V single phase and is delivered on lines 1a, 1b, and 1c. Lines 2a, 2b and 2c are coupled to ground or common. In operation, when the temperature decreases as the result of a high ice level, the thermostat 87 switches off the voltage (V) delivered on lines 1a, 1b and 1c.

Extremely low temperatures are used to achieve a

super low temperature in which the auger 54 rotated under the high torque motor 56 can shave the ice and produce super amount of ice in a small amount of time and with little water. This is achieved by the TXV 45 in conjunction with a condensing unit 24. For example, the system 10 can produce 380-500 pounds of ice per day.

The remote ice making sub-assembly 50 is designed to be compact so that it can be accommodated in a variety of locations where available space is constrained. In the exemplary embodiment, the remote ice making sub-assembly 50 has a height of approximately 29 ½ inches and a width and a depth of 10 inches. As can be appreciated, the remote ice making sub-assembly 50 can be stored under a cabinet, in a closet or on top of a counter. The housing unit 52 is made of aluminum, high temperature primer and baked on paint to protect the remote ice making sub-assembly 50 from salt water.

In the RV environment, in lieu of a water cooled condenser unit, an air cooled condenser is used. For example, a fan is substituted to cool the refrigerant with air.

Because many varying and differing embodiments may be made within the scope of the inventive concept herein

